

Institute of Mechanics

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Design of a Novel Petrov-Galerkin Finite Element Formulation for the Simulation of Plate Structures

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Abstract

In this work, within the framework of the Mindlin-Reissner plate theory, a novel Petrov-Galerkin Enhanced Assumed Strain (EAS) element formulation is developed. For this purpose, the Petrov-Galerkin method is employed, in which the test and trial functions are approximated differently. The foundation for the novel formulation is the EAS plate element by Simo and Rifai [3], as well as the Petrov-Galerkin EAS formulation for two-dimensional elasticity by Pfefferkorn and Betsch [2]. The combination of both methods is developed and investigated concerning its ability to produce a mesh-independent formulation.

Elimination of the Locking Modes

- Problem: transverse shear locking for distorted meshes
- Locking modes have polynomial form
- Conditions: patch test and orthogonality of shear stress and enhanced shear strain \rightarrow use of Legendre polynomials as basis

for interpolation matrices for the enhanced shear strain

- \blacksquare Locking can be eliminated with the right choice for ${\bf E}$ and $\tilde{{\bf E}}$
- Resulting stiffness matrix depends on nodal coordinates → strong mesh dependency expected

EAS for Plates by Simo and Rifai

Degrees of freedom and displacement vector

$$\mathbf{u} = \begin{bmatrix} w \\ \boldsymbol{\Theta} \end{bmatrix} = \begin{bmatrix} w \\ \beta_x \\ \beta_y \end{bmatrix}$$

Kinematics

- $\begin{bmatrix} \boldsymbol{\kappa} \\ \boldsymbol{\gamma} \end{bmatrix} = \begin{bmatrix} \nabla^{\mathsf{S}} \boldsymbol{\Theta} \\ \nabla w \boldsymbol{\Theta} \end{bmatrix}$
- Hu-Washizu 3-field formulation

$$\Pi_{\text{int}}(\mathbf{u}, \boldsymbol{\tau}, \boldsymbol{\gamma}) = \int_{\Omega} \frac{1}{2} \boldsymbol{\kappa}^{\mathrm{T}} \mathbf{E}_{\mathbf{B}} \boldsymbol{\kappa} \, \mathrm{d}A + \int_{\Omega} \frac{1}{2} \boldsymbol{\gamma}^{\mathrm{T}} \mathbf{E}_{\mathbf{S}} \boldsymbol{\gamma} \, \mathrm{d}A \\ + \int \left(\boldsymbol{\tau}^{\mathrm{T}} \left(\nabla w - \boldsymbol{\Theta} - \boldsymbol{\gamma} \right) \right) \, \mathrm{d}A$$

Mesh Distortion Test

Example of a simply supported rectangular plate with an constant area load



Enhancement of the shear strain

$$oldsymbol{\gamma} =
abla w - oldsymbol{\Theta} + ilde{oldsymbol{\gamma}}$$

Equilibrium

 $\delta \left(\Pi_{\mathsf{int}} + \Pi_{\mathsf{ext}} \right) \stackrel{!}{=} 0$

Petrov-Galerkin Finite Element Discretization

- Basic idea: different shape functions for test and trial functions
- Lagrangian shape functions for the test function
- Metric shape functions in skew coordinates for the trial function according to Xie et al. [4]
- Discrete enhanced shear strain and its virtual counterpart

$$\begin{split} \tilde{oldsymbol{\gamma}}_h^e(oldsymbol{\xi}) &= rac{j_0}{j} \mathbf{J}_0^{-\mathsf{T}} \tilde{\mathbf{E}}(oldsymbol{\xi}) oldsymbol{lpha} = \mathbf{H} oldsymbol{lpha} \ \delta \tilde{oldsymbol{\gamma}}_h^e(oldsymbol{\xi}) &= rac{j_0}{j} \mathbf{J}_0^{-\mathsf{T}} \mathbf{E}(oldsymbol{\xi}) \delta oldsymbol{lpha} = \mathbf{G} \delta oldsymbol{lpha} \end{split}$$

- Three nodes are shifted at angles of 45°
- Strong mesh sensitivity as expected
- Same result as ANS element of Bathe and Dvorkin [1] for regular mesh (s = 0)
- Conclusion: mesh independence cannot be achieved

References

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